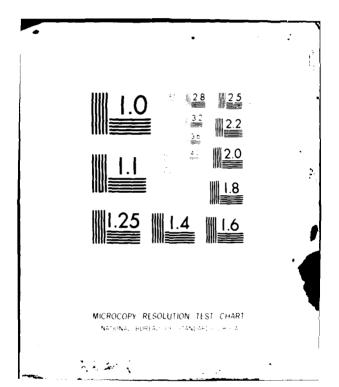
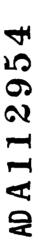
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RESEARCH AND DEVELOPMENT TECHNICAL REPORT DELET-TR-81-19

SUBTLE PERTURBATIONS AFFECTING QUARTZ RESONATORS USED FOR HIGH-STABILITY APPLICATIONS

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ELECTRONICS TECHNOLOGY & DEVICES LABORATORY

SEPTEMBER 1981

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CONTENTS

INTRODUCTION		. 1
MAGNETIC FIELDS		. 1
RESONATOR ASYMMETRIES		. 7
ELECTRODE INFLUENCES		. 29
ANGULAR MISORIENTATION		. 36
X-RAY ANGLE CHANGES		. 36
TWINNING		. 39
DYNAMIC THERMAL EFFECTS		. 43
AMBIENT VIBRATIONS		. 45
ACOUSTIC VISCOSITY		. 54
COSMIC RAYS		. 61
CONCLUSIONS		. 6 9
REFERENCES	•	. 72
FIGURES:		
1. Early Example of Quartz Synthesis	•	. 2
2. Examples of Modern Quartz Synthesis		. 3
3. Progress in Timekeeping Accuracy		. 4
 Fractional Frequency Stability of Several Types of Oscillators as Function of Sample Time 	•	. 5
5. Crystal Resonator in Magnetic Field		. 6
6. Analog Network of Resonator with Electrode Imbalance	• (. 8
7. Analog Network of Resonator in Magnetic Field	• '	. 9
8. Bisected Analog Network	, ,	. 10
9. Frequency Shifts Due to Magnetic Fields	, ,	. 11
10. Construction for Resonance and Antiresonance Frequencies	; ,	. 13

11.	Normalized Resonance Frequency Versus Coupling 14
12.	Symmetric and Asymmetric Resonators
13.	Construction for Antiresonance Frequencies with Imbalance $$. 16
14.	Wideband Spectrograph of Imbalanced Resonator 18
15.	Spectrograph Near the Second Harmonic
16.	Spectrograph Near the Fourth Harmonic
17.	Spectrograph Near the Fourteenth Harmonic
18.	Spectrograph Near the Fifteenth Harmonic
19.	Photograph of Spectrum of Balanced Resonator
20.	Photograph of Spectrum of Imbalanced Resonator
21.	Magnification of Spectrum of Figure 20
22.	Composite Resonator Structure
23.	Equivalent Network of Composite Resonator
24.	Frequency-Temperature Plots Showing Activity Dips 28
25.	Frequency-Temperature Plots of AT-Cut Resonators
26.	Apparent Angle Shift versus Mass Loading
27.	Angle Shift versus Loading for Harmonic Transitions 32
28.	Experimental Angle Shifts versus Mass Loading
29.	Apparent Angle Shift in Minutes of Arc
30.	Inflection Temperature Change with Increasing Mass 35
31.	Doubly Rotated Cuts and Zero Temperature Coefficient Loci . 37
32.	Delta Theta versus Delta Phi for AT Cuts
33.	Change of Orientation with Angle
34.	Thermoelastic Coefficient versus Theta

35.	Electrical and Optical Twinning
3 6.	Definitions Relating to Ovenized Oscillators
37.	Dynamic Thermal Effect; $\Delta T_0 = 60K \dots 46$
38.	Dynamic Thermal Effect; $\Delta T_0 = 10K \dots 47$
39.	Dynamic Thermal Effect; $\Delta T_0 = 5mK \dots 48$
40.	Dynamic Thermal Effect; $\Delta T_0 = 0.5$ and 1 mK
41.	Dynamic Thermal Effect; $\Delta T_0 = 0.5 \text{ mK} \dots 50$
42.	Ambient Accelerations: Road/Rail/Ship
4 3.	Ambient Accelerations: Aircraft/Rocket/Seismic
44.	Seismic Noise of the Earth's Surface
45.	Conventional and Ring-Supported Resonators
46.	Ring-Supported and Grooved Crystal Resonators
4 7.	Definitions of Ring-Supported Resonator Parameters 57
4 8.	Acceleration-Frequency Effects of X-Axis Direction for Sample No. T-4 58
49.	Acceleration-Frequency Effects of X-Axis Direction for Sample No. B-4 59
50.	Experimental Values of Q versus Frequency for Quartz 60
51.	Time Constant, τ_1 for AT-Cut Quartz Plates as a Function of Diameter-Thickness Ratio, $\phi a/t$, Average Values 62
52.	Time Constant, τ_1 , for AT-Cut Quartz Plates in Vacuum as a Function of Diameter-Thickness Ratio, $\phi a/t$ 63
53.	Cosmic Ray Flux versus Particle Energy at Sea Level 64
54.	Flux versus Energy in MeV at Sea Level 65
55.	Flux versus Energy at 3 and 10 km Height 66
56.	Frequency Shift for Photons
57.	Frequency Shift for Neutrons

58.	Frequency Stability versus Year	′(
59.	Progression in Crystal Technology	, :

INTRODUCTION

The year 1980 marks the centenary of the discovery of piezoelectricity by the brothers Curie, and the fifty-ninth year of the quartz oscillator. The past thirty years or so have seen the development of cultured quartz from coin-sized pieces, shown in Fig. 1, to large, multi-kilogram crystals that constitute the bulk of the usage today (Fig. 2). Since the introduction of the quartz crystal-controlled oscillator by Cady, each decade of time has seen approximately one order of magnitude frequency stability improvement. Surrounding the development of high-stability oscillators incorporating crystal vibrator control has been the on-going interplay between improvements in resonator technology and causes of ever-smaller perturbations in resonator frequency and equivalent circuit parameters. 1-42* Progress in timekeeping is shown on a broad time scale in Fig. 3, and a comparison of crystal resonator performance with atomic and molecular frequency standards is shown in Fig. 4; both Figures are adapted from Reference 2.* Further improvements will depend upon less-obvious factors than have been considered to date. This technical report treats a number of such topics.

MAGNETIC FIELDS

Quartz, and other high purity dielectric crystals, can be considered to be nonmagnetic for our purposes here. A vibrating crystal with metallic, nonmagnetic electrodes is, however, affected by external magnetic fields. Eddy currents are produced in the electrodes by their motions in the fields; these change the critical frequencies of the vibrator, and lower its Q. The situation considered is that of a resonator vibrating in a pure shear mode with applied static magnetic field parallel to plate thickness as depicted in Fig. 5. The resonator leads are attached to an active network comprising the remainder of the oscillator, and the combination adjusted in frequency so that the vibrator operates at zero reactance at all times.

For a pure shear mode, the retarding force density at the plate surfaces is

$$- \mathbf{j} \omega \sigma \mathbf{B}^2 \mathbf{u}, \tag{1}$$

with σ the electrical conductivity of the electrode, and u the mechanical displacement at the surfaces. Electrodes of thickness h' will produce resistances of

$$R = A B^2 h' (2)$$

for areas of value A. Using conventional notation 3 , this leads to an expression for the imput admittance of the single mode vibrator, including the effects of electrode inertia 4 :

^{*}See list of references beginning on page 72.

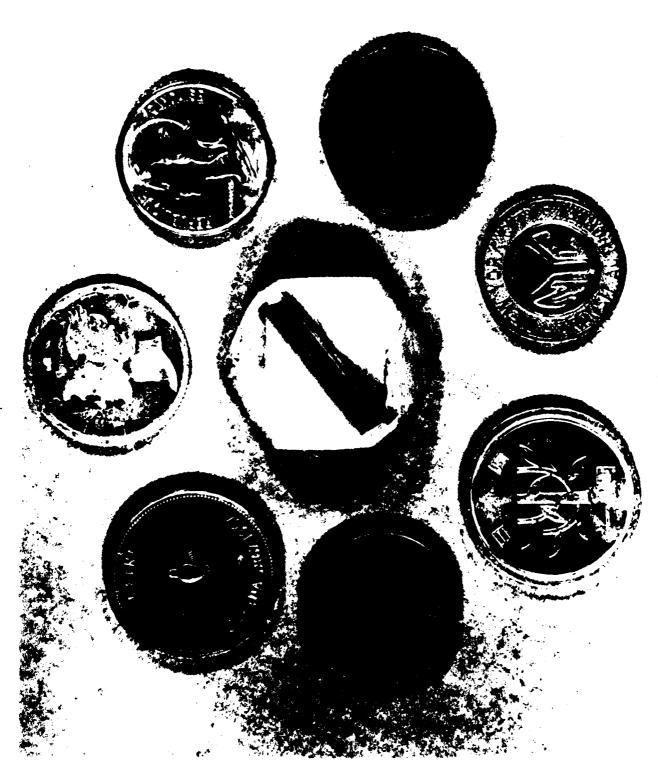
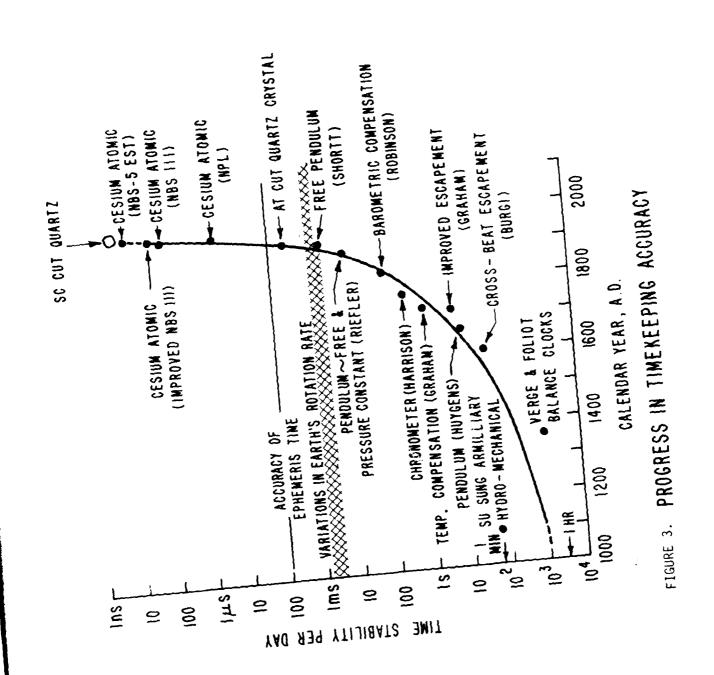


FIGURE 1. Early Example of Quartz Synthesis



FIGURE 2. Examples of Modern Quartz Synthesis



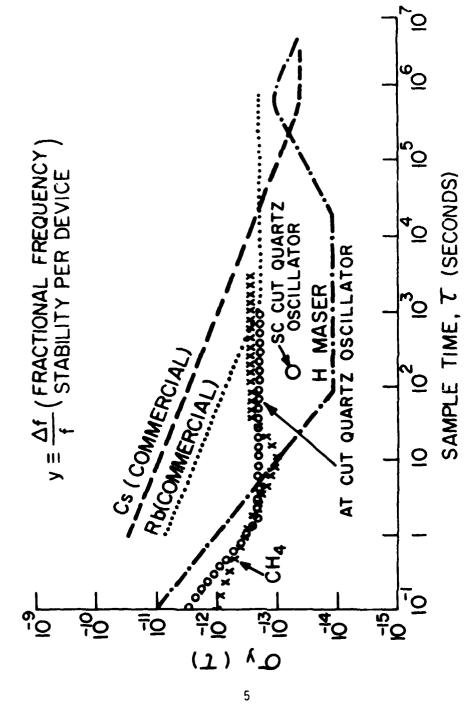
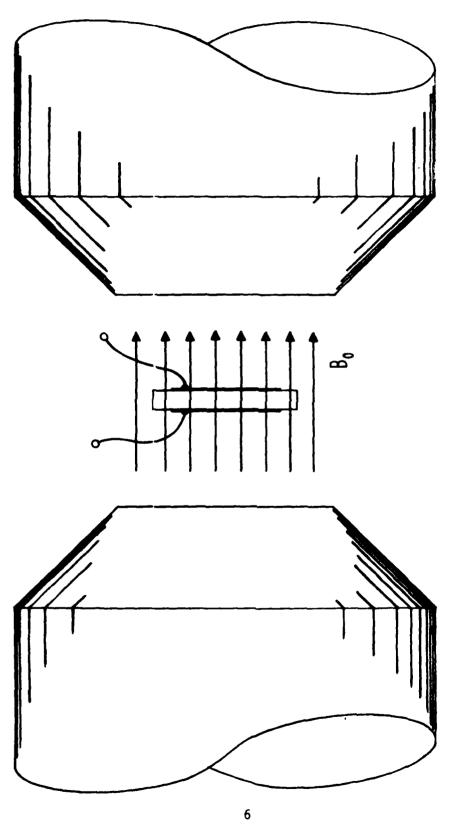


FIGURE 4. FRACTIONAL FREQUENCY STABILITY OF SEVERAL TYPES OF OSCILLATORS AS FUNCTION OF SAMPLE TIME.



CRYSTAL RESONATOR IN MAGNETIC FIELD FIGURE 5.

$$\frac{\text{Yin} = j \omega C_{0} + j \omega C_{0}}{\{(X/(k' \tan X) - 1 - X^{2} \mu/k^{2}) + (jXRY_{0}/k')\}^{-1}}.$$
(3)

Resonance and antiresonance frequencies are determined from the zeros of the susceptance function as the roots of

$$(X-(k^2 + \mu X^2) \tan X) \cdot (1- \mu X \tan X)$$

+ $X \cdot (R Y_0)^2 \cdot \tan^2 X = 0.$ (4)

When R=0, the first expression in brackets yields the condition for resonance, and the second that for antiresonance. From the expression for Y_{in} the network in Fig. 6 is obtained, with the electrode inertia represented by the inductances

$$L_1 = L_2 = L = \rho h \mu A.$$
 (5)

The network components are labeled with their physically corresponding effects. When R is not zero, the inductances are replaced by series R-L combinations, and when electrode mass-loading is neglected, the circuit appears as given in Fig. 7. Figure 8 is the bisected version of Fig. 7 with massless electrodes (μ =0).

Departures from the resonance frequency as function of R are found by perturbation, with RY $_{\rm O}$ the small parameter. The expression for frequency shift is

$$\Delta f/f \approx - (2/\pi)^2 (RY_0)^2,$$
 (6)

with
$$RY_0 = \sigma h^{\alpha} B^2 / (\rho v)$$
, (7)

the quartz mass density, and v the acoustic velocity of the mode in question. It is seen from (6) and (7) that the frequency shift is quartic in the magnetic field strength. Figure 9 gives frequency shift values for an AT cut having 10,000 Å copper electrodes; only for fields about ten thousand times larger than that of the earth's (the first entry) is the effect perceptible.

This analysis has been extended to the multimode case, and to the situation of a lateral magnetic field, with corresponding network representations.

RESONATOR ASYMMETRIES

In treatments of resonator electroding, it is usually tacitly assumed that both electrodes are of equal size and thickness. The practical reality is usually just the opposite. It is very difficult to guarantee that the thicknesses are equal in the first place; to make matters worse, final frequency adustment is usually made with a burst of material applied to a

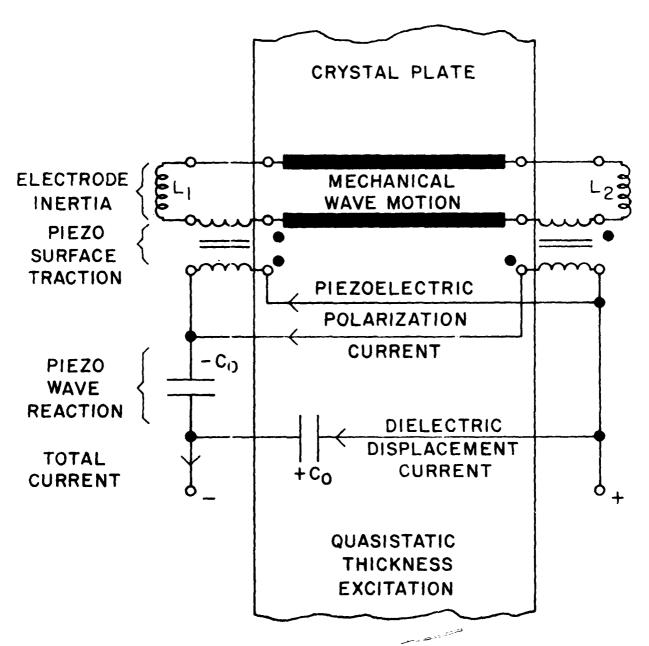


FIGURE 6. Analog Network of Resonator with Electrode Imbalance

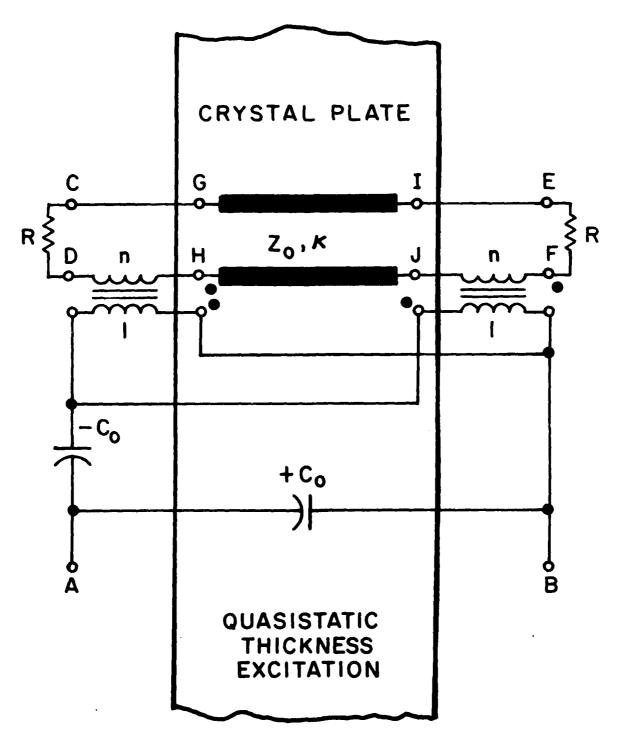


FIGURE 7. Analog Network of Resonator in Magnetic Field

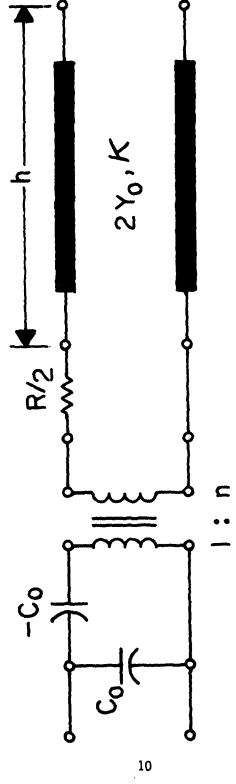


FIGURE 8. Bisected Analog Network

$$\Delta F/F \simeq -(\frac{2}{\pi})^2 * R^2 \gamma_0^2$$

RYO = σ H' B²/(ρ v)

	KILO-	OERSTED	0,58 (-3)	~	~	10	30
=	AMPERE/	METER	46.2	(4+) 6'2	2,4 (+5)	(2+) 6'2	2.4 (+6)
ΔF/F	AT CUT	1µM CU ELECTRODES	-2,2 (-28)	-1,9 (-15)	-1,5 (-13)	-1,9 (-11)	-1.5 (-9)
	KILO-	GAUSS	0,58 (-3)	7	3	10	û
B		TESLA	5,8 (-5)	0.1	0,3	7	M

FIGURE 9. Frequency Shifts Due to Magnetic Fields

single side. Add to this situation the facts that registration of both electrodes is never perfect, that the electrode tabs are asymmetrically disposed, and that sometimes a larger electrode is used on one side to improve statistical unif rmity of the motional capacitance by compensating for misregistration, then it is seen that symmetry of the plate resonator about the mid-plane is seldom, if ever, achieved.

The single-mode case is treated, by way of example, for a simple electrode mass imbalance in the one-dimensional approximation. The equivalent network is that of Fig. 6 with the mechanical transmission line ports attached to unequal transmission lines representing the thick electrodes. When the electrodes are thin enough for propagation of the acoustic waves within the electrodes to be neglected, the transmission lines are replaced with lumped inductances; the mass imbalance results in unequal inductance values L_1 and L_2 , as shown, and prevents the network from being bisected, as in Fig. 8. When bisection can take place, the transmission line representing the crystal plate of thickness 2h is replaced by one of half this value and the construction for the critical frequencies is that given in Fig. 10; the formulas are those of the bracketed terms in (4) separately set to zero. Figure 11 gives the normalized frequency variable

$$X_{RO}^{(M)} = (\pi/2) (f_{RO}^{(M)}/f_{AO}^{(1)})$$

versus piezoelectric coupling k for the first three harmonics M=1, 3,and 5. An AT-cut resonator with symmetrical electrodes of equal thickness produced the mode spectrograph at the top of Fig. 12. Strong responses are found at the odd harmonics, and nothing is visible at the even harmonics.

It is appropriate now to quote from Cady¹, p. 308:

"There are circumstances under which a plate may conceivably vibrate in a thickness mode at or close to an even harmonic frequency. The shape, size and location of the electrodes may be such as to produce a driving field in the plate that varies in the direction of the thickness. Or the plate may be twinned or have other defects such that the excitation is not uniform. Finally, the plate may be in contact with an electrode of considerable mass, so that in effect one has a composite resonator."

When the electrodes are of unequal mass, and circuit bisection is not possible, the construction becomes that of Fig. 13, where the critical frequencies are determined by intersection with a tangent function of argument 2X instead of X. The expression for input impedance is

$$Z_{in} = (1/j\omega C_{o}) \cdot \{1 - \frac{k^{2} (\tan X/\dot{X})}{(1 - \bar{\mu}X \tan X) + \frac{\Delta^{2}X^{2}}{(1 + \bar{\mu}X \cot X)}}\},$$
 (8)

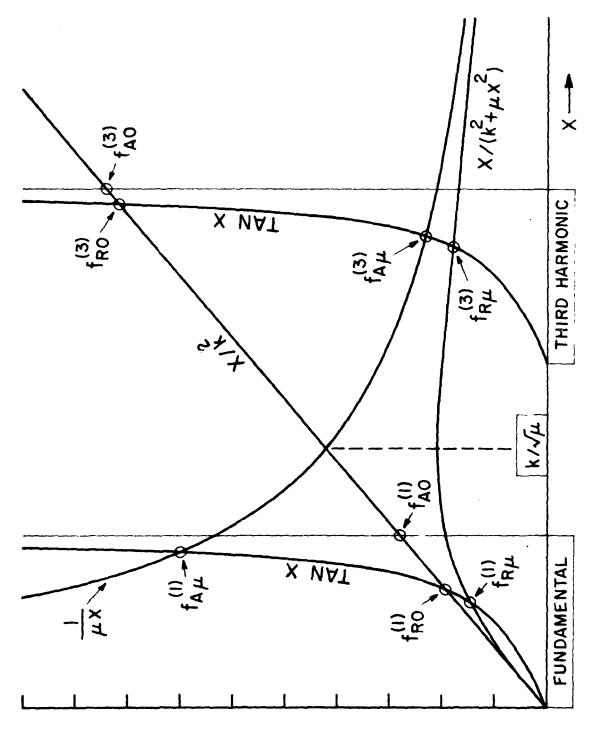


FIGURE 10. Construction for Resonance and Antiresonance Frequencies

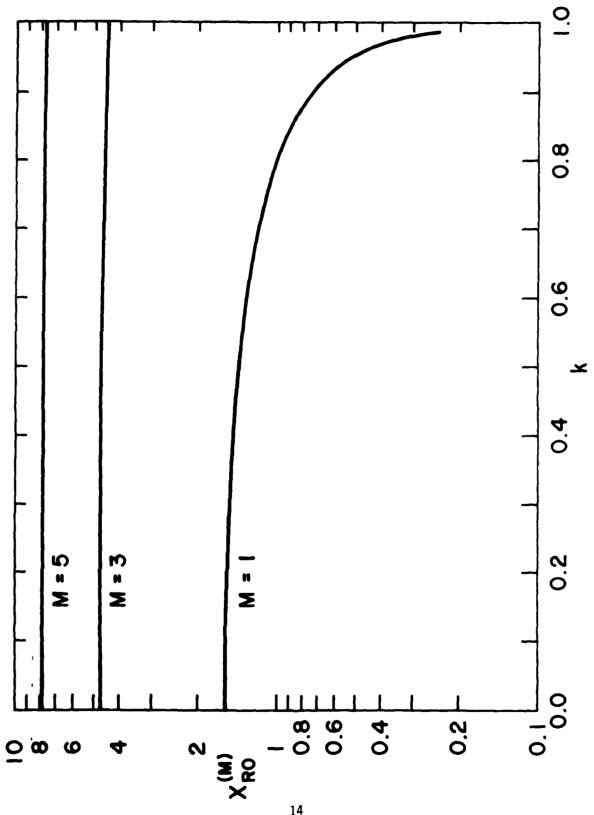


FIGURE 11. Normalized Resonance Frequency versus Coupling

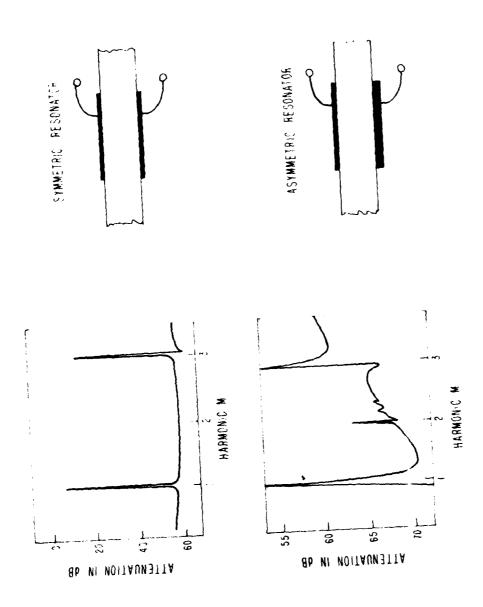


FIGURE 12. Symmetric and Asymmetric Resonators

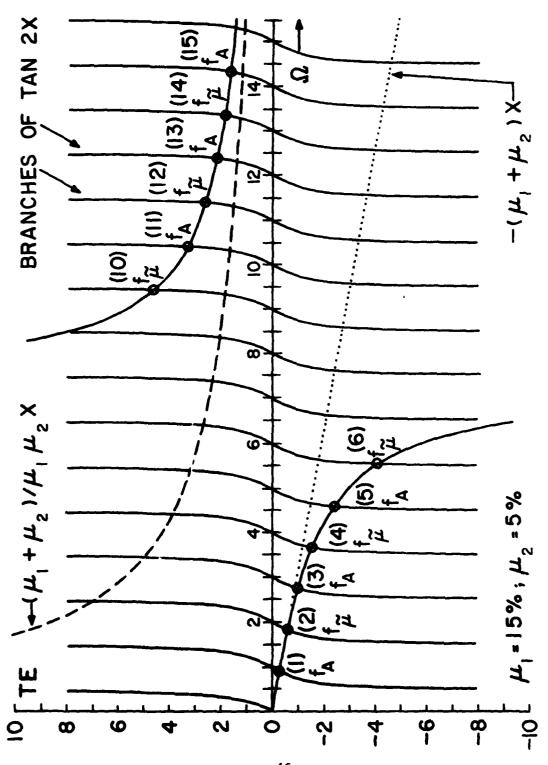


FIGURE 13. Construction for Antiresonant Frequencies with Imbalance

where $\tilde{f}=(\mu_1+\mu_2)/2$ and $\Lambda=(\mu_1-\mu_2)/2$. Equation (8) has been written in terms of argument X for the tangent and cotangent functions. The anti-resonant frequencies (denoted f (M) for M odd, and f (M) for M even) are found from the vanishing of the bracketed term in (8) as roots of

$$\tan 2X = \frac{-(\mu_1 + \mu_2) X}{(1 - \mu_1 - \mu_2 X^2)}.$$
 (9)

These are shown in Fig. 13 along with the high and low frequency asymptotes of the right hand side of (9). The roots for the resonant frequencies are found from the zeros of (8).

The even-harmonic resonances produced by the unequal mass-loadings on the crystal plate surfaces ($\mu_1 \neq \mu_2$) are very sharp compared to the odd harmonics as may be seen from Fig. 14, which is drawn for a case with large piezo-coupling and large mass asymmetry in order to widen the resonance regions of both the even and odd harmonics. It is thus possible to use such resonances for stabilizing the frequency of an oscillator - at least in principle. Figure 15 shows a blow-up of the second harmonic of the case of Fig. 14; Fig. 16 shows the fourth harmonic, and Fig. 17 shows the fourteenth harmonic. Compare the width of the resonance in Fig. 17 with that of Fig. 18 showing the fifteenth harmonic. In general, for large M, the normalized frequency of response approaches (M-1).

When the resonator shown in the top of Fig. 12 was overcoated with electrode material on one side only, the mode spectrum shown in the bottom of the figure was produced. The second harmon c becomes present due to the imbalance, and it is sharp; however, it is also very weak. Figure 19 shows a photograph of the spectrograph given in the top of Fig. 12; Fig. 20 is a photograph of the spectrograph for the case of mass imbalance to the same scale as Fig. 20; Fig. 21 shows a magnified view of the photograph of Fig. 20; it corresponds to the lower drawing in Fig. 12.

Another unsymmetrical structure that leads to even harmonics is the composite resonator; (see Reference 5 for additional references). This structure is shown in Fig. 22, with the equivalent network given in Fig. 23. Here the quartz substrate must be represented by a transmission line because of its thickness; as it shrinks in size the situation of Fig. 6 with inductors is approached.

Apart from producing even harmonics, the presence of asymmetry in resonators (due to unequal electrodes, bevelling, contouring, etc.) is very probably a major cause of "activity dips," or "bandbreaks." These are illustrated in Fig. 24, which is taken from Reference 8. Both the frequency versus temperature and the conductance versus temperature curves suffer anomalous dips as the interfering mode's influence is manifest at a particular temperature. The second harmonic mode "divided by two" via the nonlinear elastic constants can produce dips at the fundamental.

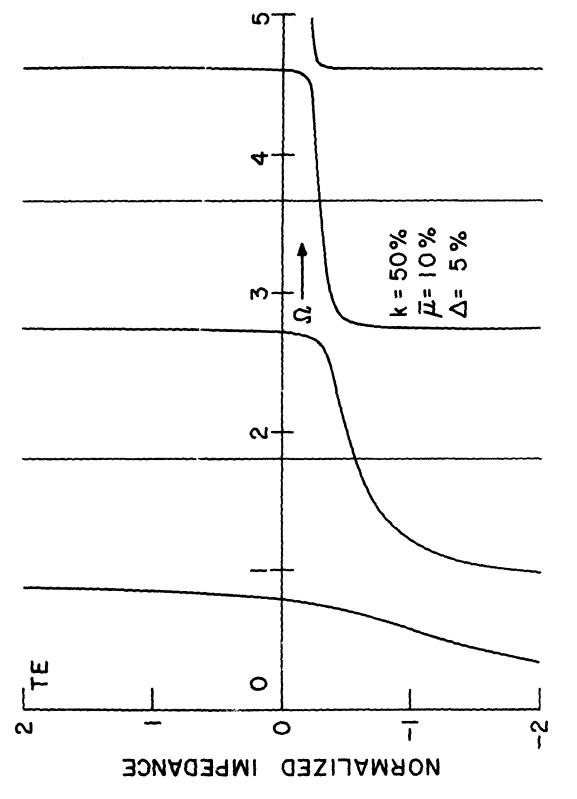


FIGURE 14. Wideband Spectrograph of Imbalanced Resonator

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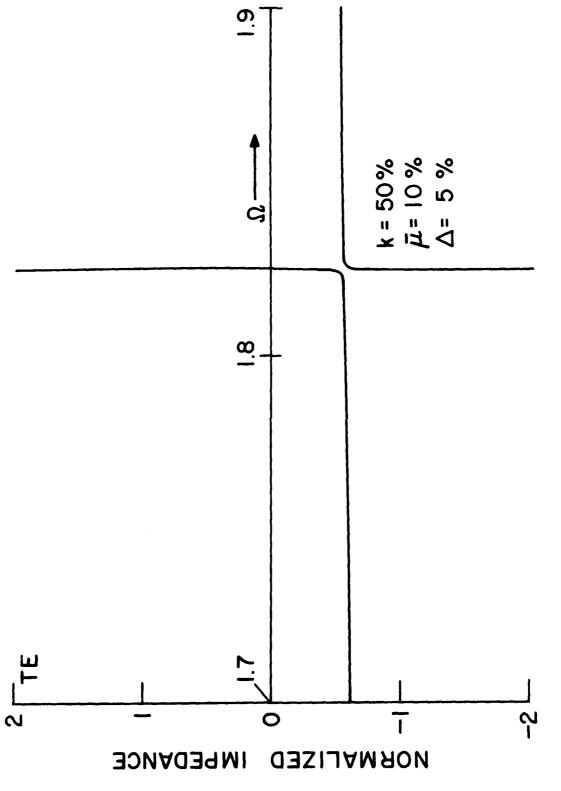
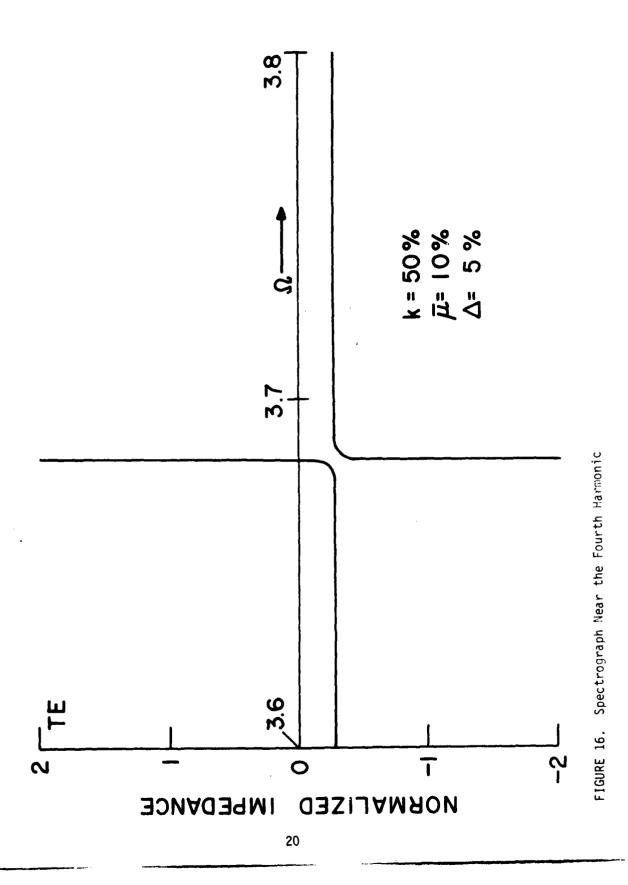


FIGURE 15. Spectrograph Near the Second Harmonic





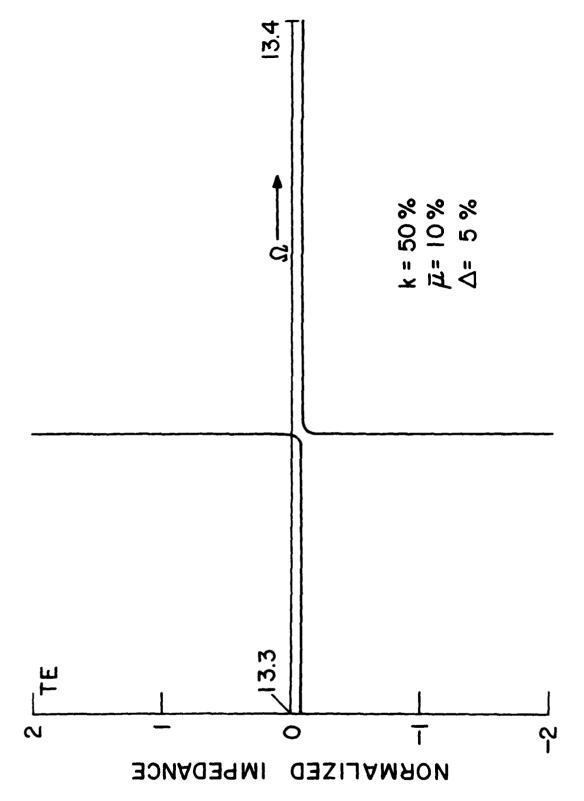
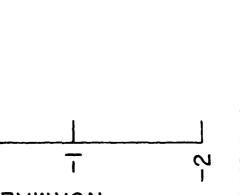
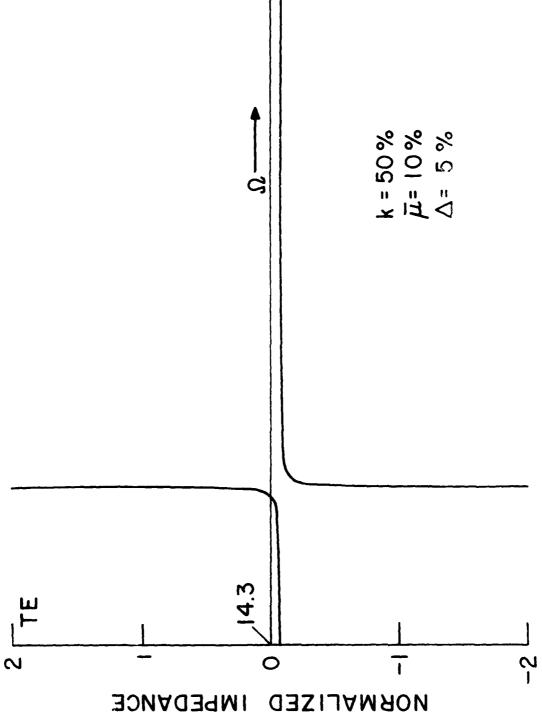


FIGURE 17. Spectingraph Near the Fourteenth Harmonic





14.4

FIGURE 18. Spectrograph Near the Fifteenth Harmonic

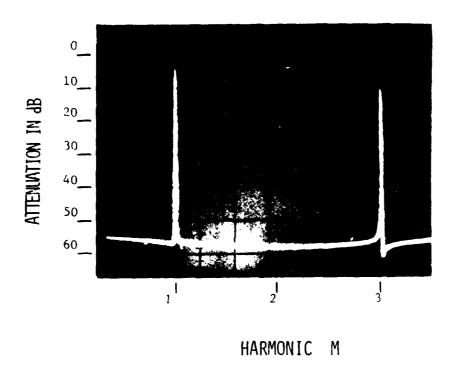


FIGURE 19. Photograph of Spectrum of Balanced Resonator

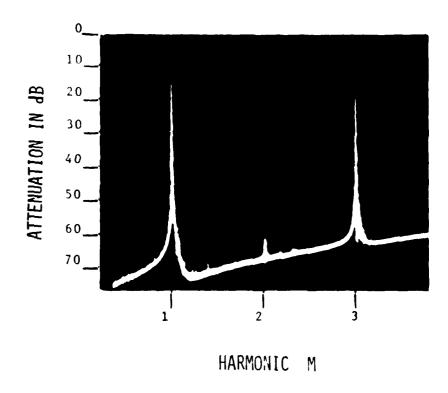


FIGURE 20. Photograph of Spectrum of Imbalanced Resonator

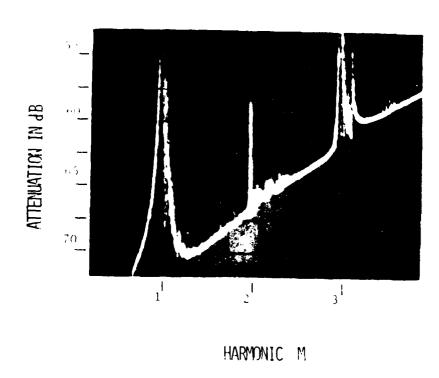


FIGURE 21. Magnification of Spectrum of Figure 20

THIN-FILM LAYER OF HIGH-COUPLING PIEZOELECTRIC CRYSTAL DRIVES THE COMPOSITE STRUCTURE. 7

SUBSTRATE QUARTZ ELECTRODES -

FIGURE 22. COMPOSITE RESONATOR STRUCTURE

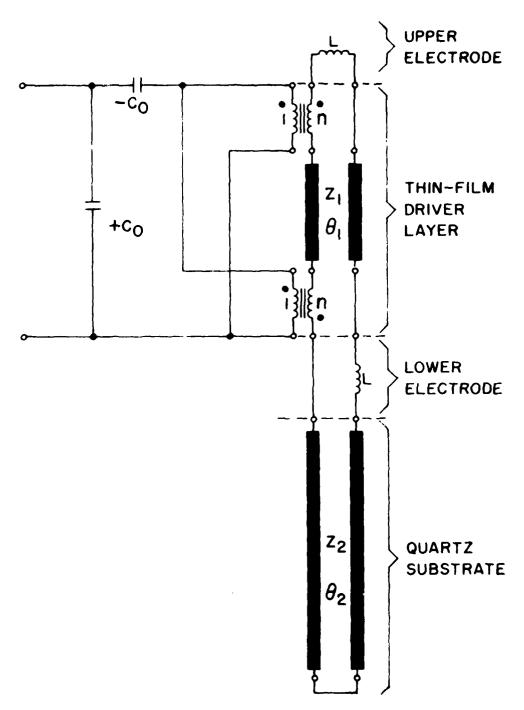


FIGURE 23. EQUIVALENT NETWORK
OF COMPOSITE RESONATOR

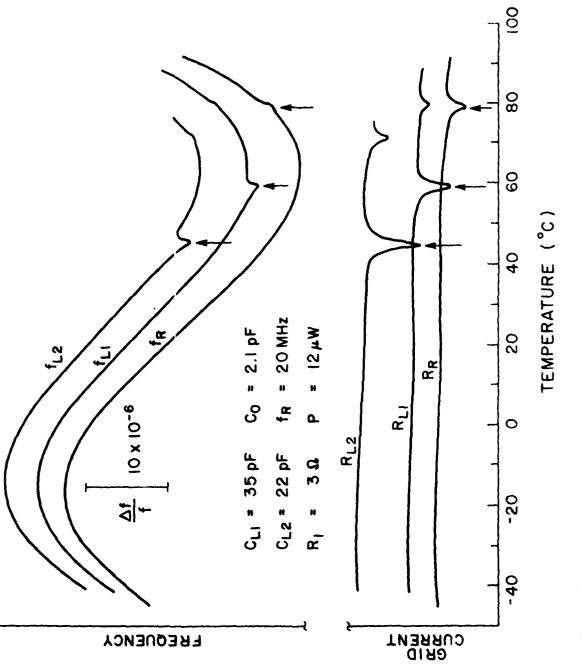


FIGURE 24. Frequency-Temperature Plots Showing Activity Dips

ELECTRODE INFLUENCES

It is well-known that the frequency-temperature (f-T) curve of an AT plate resonator measured sufficiently slowly assumes a cubic shape, as depicted in Fig. 25. Small changes in orientation angle θ , for rotation about the crystallographic X_1 axis, produce slope changes at the inflection point of -5.08 x $10^{-6}/\text{K}$, θ . A variety of other influences lead to nonnegligible slope variations as well. Figure 25 shows the effect of placing a load capacitor in series with the vibrator to produce frequency f_{L} , as is often done for frequency trimming. The slope shift reduces the peak-to-peak frequency deviation and must be taken into account in design of temperature compensated crystal oscillators (TCXOs); it comes about because of the temperature coefficient of the piezoelectric coupling factor. A similar effect occurs when a resonator is operated at different harmonics, and for the same reason.

The mass of, and intrinsic stress in electrode coatings also affects the f-T curve. Some time ago the effect of equal electrode coatings was demonstrated in regard to frequency shift and rotation of the static frequency-temperature curve of a resonator. $^{11-13}$ The calculations dealt only with the influence of inertial mass; no elastic properties were assumed for the electrodes, nor were they presumed to apply any stresses to the crystal. This was done so that the inertial influence of the electrode could be subtracted from elastic and stress effects. Results from the one dimensional theory for the change in apparent orientation angle are given in Fig. 26 and Fig. 27 as function of harmonic number M and mass loading μ .

A comparison of apparent orientation angle shifts as function of mass loading and harmonic number is given in Fig. 23 for AT plates of 5MHz fundamental frequency, and plate/electrode diameters of 14 and 6 mm, respectively, coated with evaporated aluminum, sputtered chromium, and sputtered indium. The electrodes are of equal thicknesses. It is seen that Al and In have slopes that agree with inertial mass theory, but are displaced; the curves for Cr are consistent among themselves, but differ in sign from the theory. The results in Fig. 28 indicate the importance of electrode material, deposition method, and conditions of deposition on the behavior of the finished resonator. Figure 29 gives the experimental data for aluminum evaporated electrodes as function of μ and M. Figure 30 shows the decrease in inflection temperature with increasing mass of evaporated aluminum, for resonators with the parameters described above.

It should be borne in mind that stresses in the electrodes can also produce activity dips, even when the electrodes are of equal mass. Consequently, the practice of furnishing AT quartz plates with a thin layer of Ni or Cr to insure adhesion of gold, e.g., ought to be re-examined in view of the stress levels produced by these metals, both intrinsically, and as a function of temperature.

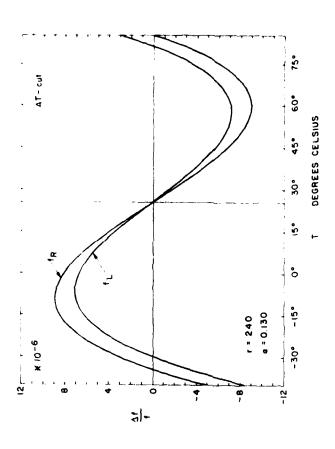


FIGURE 25. Frequency-Temperature Plots of AT-Cut Resonators



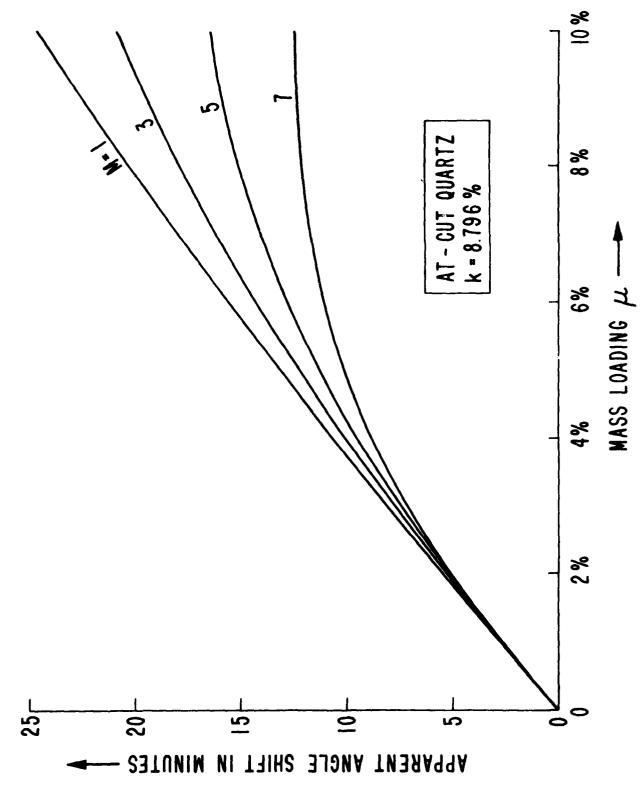


FIGURE 26. Apparent Angle Shift versus Mass Loading

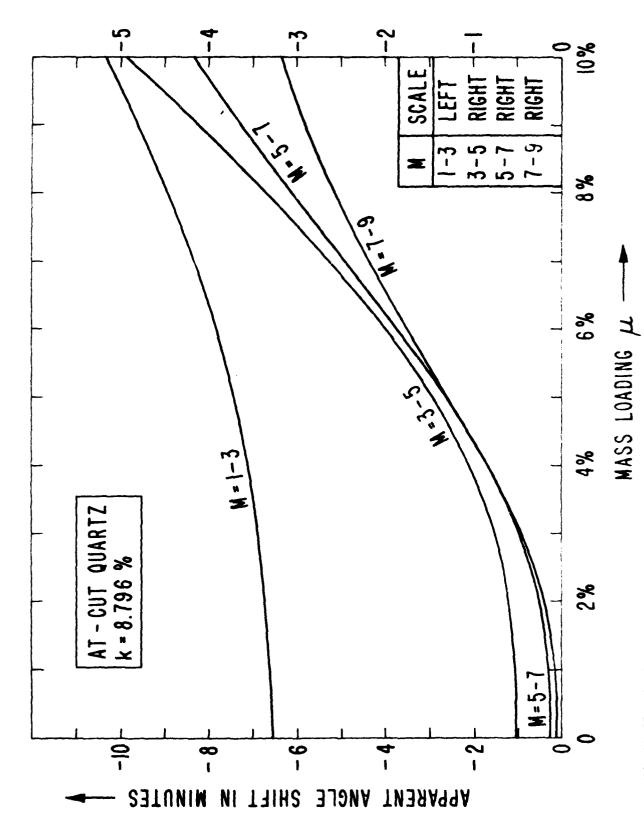


FIGURE 27. Angle Shift versus Loading for Marmonic Transitions

Li

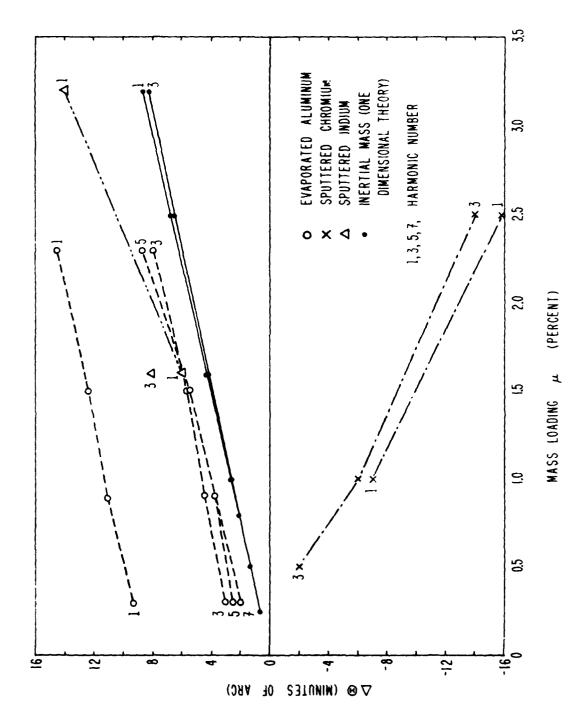


FIGURE 28. Experimental Angle Shifts versus Mass Loading

HARMONIC NUMBER

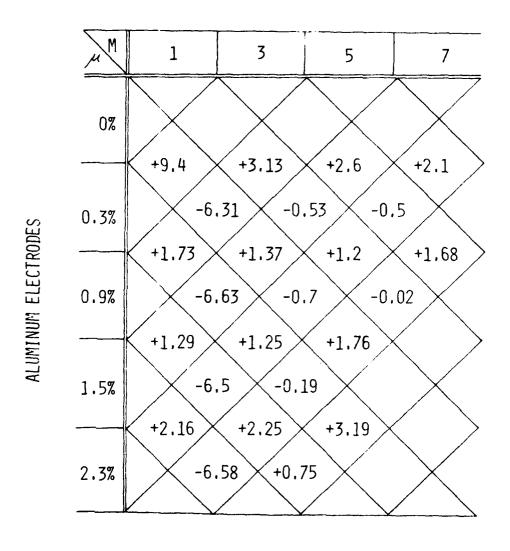


FIGURE 29. APPARENT ANGLE SHIFT IN MINUTES OF ARC

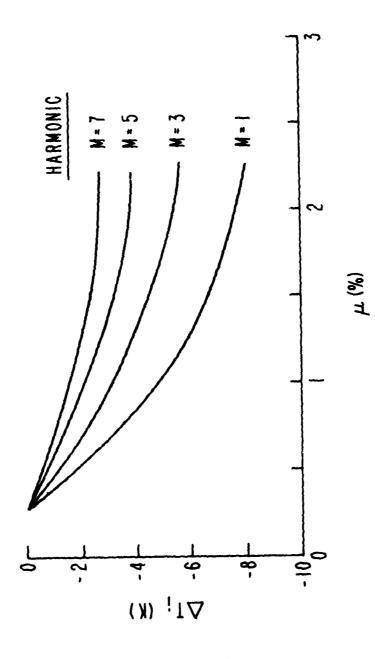


FIGURE 30. INFLECTION TEMPERATURE CHANGE WITH INCREASING MASS

١,

ANGULAR MISORIENTATION

Subtle influences on the static frequency-temperature curve were briefly discussed in the previous section in terms of apparent orientation angle changes in θ , for AT resonators. The most general class of thickness mode plate resonators is the family of doubly rotated cuts. ¹⁴ These are compared in Fig. 31 with singly rotated cuts, of which the AT cut is the most often used. Also shown in the figure are the loci of zero temperature coefficient, and the angular coordinates of popular quartz cuts. The line ϕ = 0 is a line of symmetry, so that singly rotated cuts are only quadratically sensitive to misorientations in ϕ . For the AT cut, misorientations $\Delta \Phi$ may be compensated for by changes $\Delta \theta$, obtained from

$$\Delta\theta = -1.77 \times 10^{-3} (\Delta\phi)^2$$
,

with $\Delta \phi$ in minutes of arc, and $\Delta \theta$ in seconds of arc; this relationship is portrayed in Fig. 32.

X-RAY ANGLE CHANGES15

The angles locating the X-ray planes in a crystal are functions of temperature and external stress. For example, the plane at 38° 13' used for orienting AT cuts is only found at this angle at a certain temperature. For high precision applications the angle shift with temperature is not negligible. We give an approximate calculation to establish the order of magnitude of the angle change.

Consider a rotated-Y-cut plate of crystal in the rhombic, tetragonal, trigonal, hexagonal, or cubic systems. (These have no off-diagonal thermoelastic coefficients.) At temperature T the plate angle will be determined from

$$tan \in = (X_2/X_3). \tag{10}$$

With a change in temperature to $T + \Delta T_{\star}$ the axes become, in the linear approximation

$$X_2 = X_2 (1 + \alpha_{22} \Delta T)$$

 $X_3 = X_3 (1 + \alpha_{33} \Delta T)$ (11)

We are thus led to a relation between $\Delta\theta$ and ΔT :

$$tan \Delta \theta = \{(A-1) tan \theta / (1 + A tan^2 \theta)\}$$
 (12)

where

$$A = (1 + \alpha_{22} \Delta T)/(1 + \alpha_{33} \Delta T). \tag{13}$$

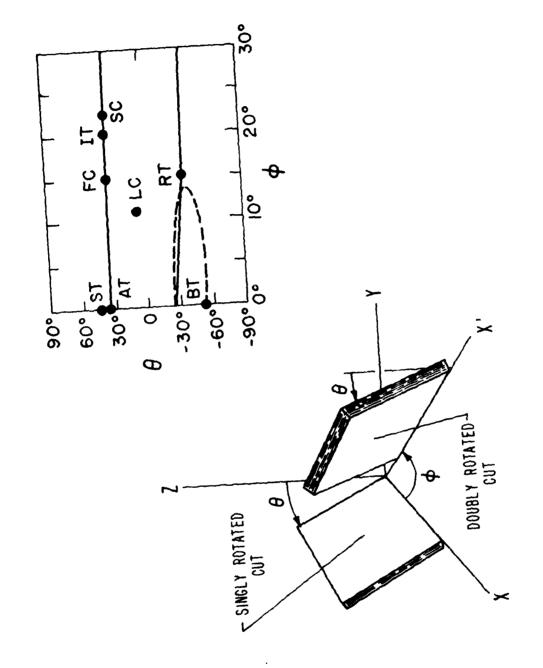


FIGURE 31. Doubly Rotated Cuts and Zero Temperature Coefficient Loci

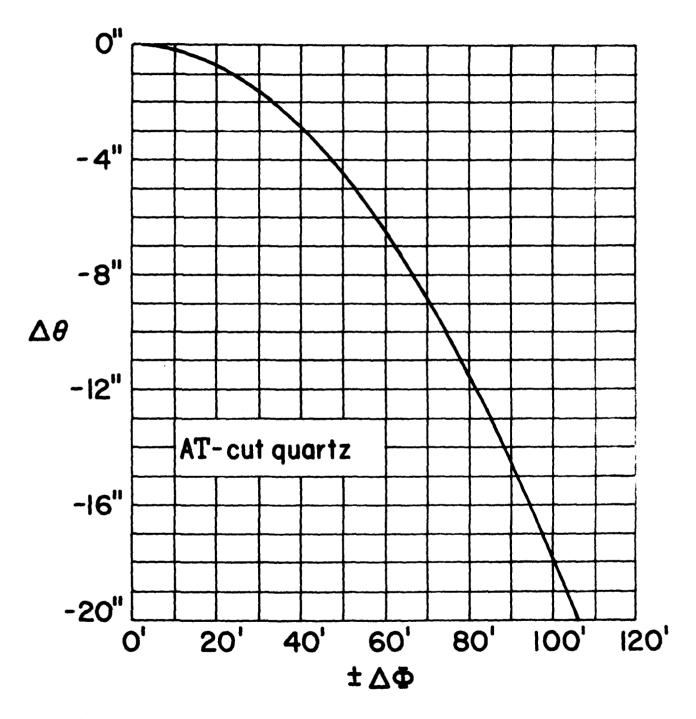


FIGURE 32. Delta Theta Versus Delta Phi for AT Cuts

As a consequence of (12) and (13) we have

$$\Delta\theta \approx \frac{1}{2} \left(\alpha_{11} - \alpha_{33} \right) \cdot \Delta T \cdot \sin 2\theta. \tag{14}$$

Figure 33 gives the X-ray angle change with temperature; beyond about 10 kelvins the linear approximation is not accurate except as to order of magnitude. For the tetragonal, trigonal and hexagonal systems, since $\alpha_{22} = \alpha_{11}$, the result is independent of angle ϕ . In particular, for the upper zero temperature locus for thickness mode plates θ is approximately constant and the result holds therefore for the AT, FC, IT, and SC cuts (cf. Fig. 31). Relation (14) shows that $\Delta\theta$ depends on the difference between α_{11} and α_{33} for rotated-Y-cuts (or doubly rotated cuts when $\alpha_{22} = \alpha_{11}$). For the cubic system this difference is zero, so $\Delta\theta$ is always zero in this system. For quartz, the effective value of α in the thickness direction of rotated-Y-cut plates is shown in Fig. 34; it is given by

$$\alpha = \alpha_{11} \cos^2 \theta + \alpha_{33} \sin^2 \theta; \qquad (15)$$

for the Y cut, $\alpha = \alpha_{2}$ = $\alpha_{11} = 13.72 \times 10^{-6}$ /K, while for the Z cut ($\theta = \frac{1}{2}90$), $\alpha = \alpha_{33} = 7.48 \times 10^{-6}$ /K.

TWINNING

The types of twinning that may occur in quartz are shown in Fig. 35 along with the changes in axes and handedness. 16,1 We briefly explore the effects of twinning on the piezoelectric constants of crystals in Class 18, which includes quartz. For the three types of twinning considered, the operations involved are

(a) Dauphiné: $X_1 \rightarrow -X_1$; $X_2 \rightarrow -X_2$; $X_3 \rightarrow +X_3$.

$$e_{11} \rightarrow -e_{11}$$
; $e_{14} \rightarrow +e_{1}$.

(b) Brazil: $X_1 \rightarrow -X_1$; $X_2 \rightarrow +X_2$; $X_1 \rightarrow +X_3$.

$$e_{11} \rightarrow -e_{11}$$
; $e_{14} \rightarrow -e_{14}$.

(c) Combined: $X_1 \rightarrow +X_1$; $X_2 \rightarrow -X_2$; $X_3 \rightarrow +X_3$. $e_{11} \rightarrow +e_{11}$; $e_{14} \rightarrow -e_{14}$.

Consider e^2_{26} , the piezoelectric constant driving the pure shear mode in rotated-Y-cuts of quartz:

Untwinned: $e_{26} = (e_{11} \cos \theta + e_{14} \sin \theta) \cdot (-\cos \theta)$.

- (a) Dauphine: $e_{26}^* = (-e_{11} \cos \theta + e_{14} \sin \theta) \cdot (-\cos \theta)$.
- (b) Brazil: $e_{26}^2 = (e_{11} \cos \theta + e_{14} \sin \theta) \cdot (+\cos \theta)$.
- (c) Combined: $e_{26} = (e_{11} \cos \theta e_{14} \sin \theta) \cdot (-\cos \theta)$.

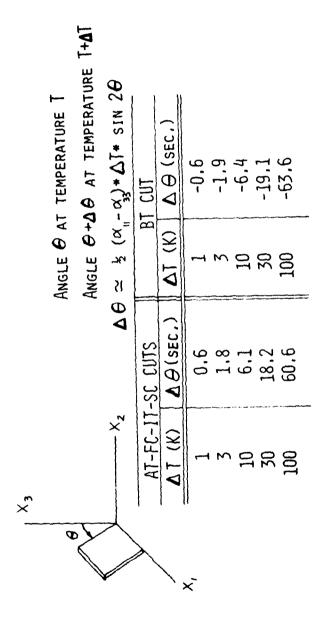
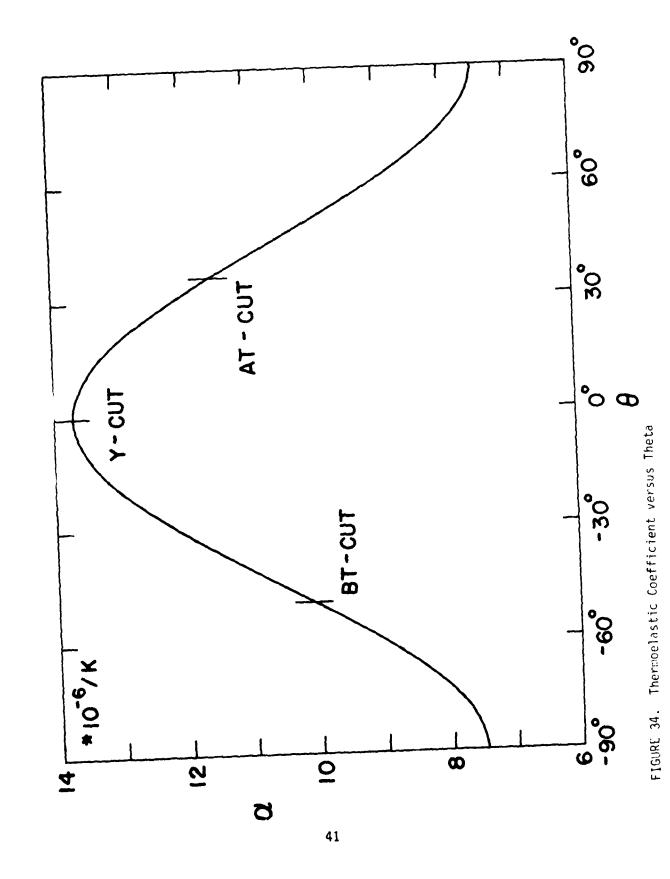


FIGURE 33. Change of Orientation with Angle



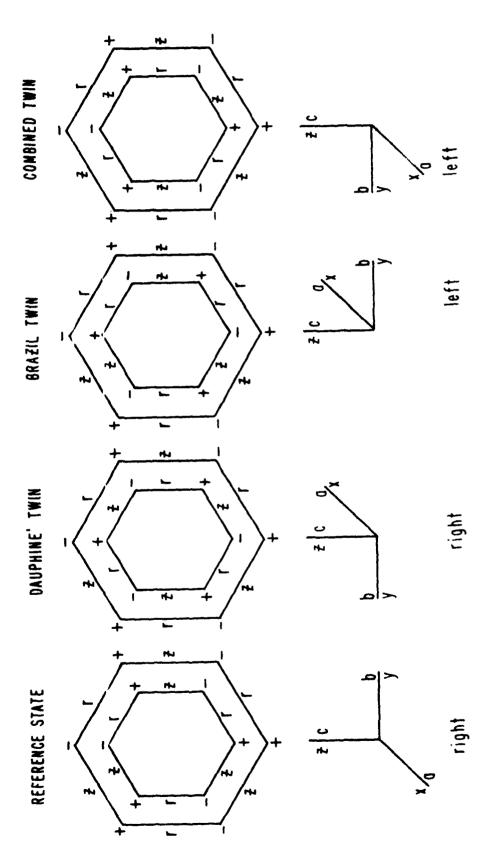


FIGURE 35. Electrical and Optical Ewinning

Consider further a plate of unity area, with a fraction "a" of this area twinned. For simplicity, assume the twinning is complete throughout the thickness, and that the distribution of motion is uniform even though twinned (although we keep in mind Cady's remark, above). Then the composite e^2_{26} should be:

(a) Dauphiné:

$$e'_{26} = ((1-2a) e_{11} \cos\theta + e_{14} \sin\theta) \cdot (-\cos\theta).$$

(b) Brazil:

$$e'_{26} = (1-2a) e_{11} \cos\theta + (1-2a) \sin\theta$$
 ' (-\cos\theta).

(c) Combined:

$$e_{26}^{\prime} = (e_{11} \cos \theta + (1-2a) e_{14} \sin \theta) \cdot (-\cos \theta).$$

In the case of Brazil twinning:

$$e_{26}^{*}$$
 (twinned) = (1-2a) e_{26}^{*} (untwinned). (16)

Equation (16), when written in terms of the resonator capacitance ratio becomes:

$$r \text{ (twinned)} = r \text{ (untwinned)} / (1-2a)^2. \tag{17}$$

DYNAMIC THERMAL EFFECTS

It has been known for many years that abrupt temperature changes produce frequency changes in quartz vibrators that are unpredicted by static f-T characteristics such as those of Fig. 25. The dynamic thermal effect has been given a phenomenological explanation by a modification of the cubic curve to include time-dependent terms that take into account thermal gradients in the resonator and support structure. The classical f-T equation

$$\frac{\Delta f}{f} = a_0 \Delta T + b_0 \Delta T + c_0 \Delta T^3$$
 (18)

is augmented by the addition of the term

$$(\hat{a} \Delta T + \hat{a}) \cdot dT/dt \tag{19}$$

A temperature ramp (dT/dt = constant) applied to the resonator both rotates the f-T curve, through a, and translates it, by means of a. Figure 36 defines the desired reference temperature (which may be at either zero-slope point) at which the oven ought to be set, along with the actual oven setting point achieved, and the oven cycling range. The cycling range is

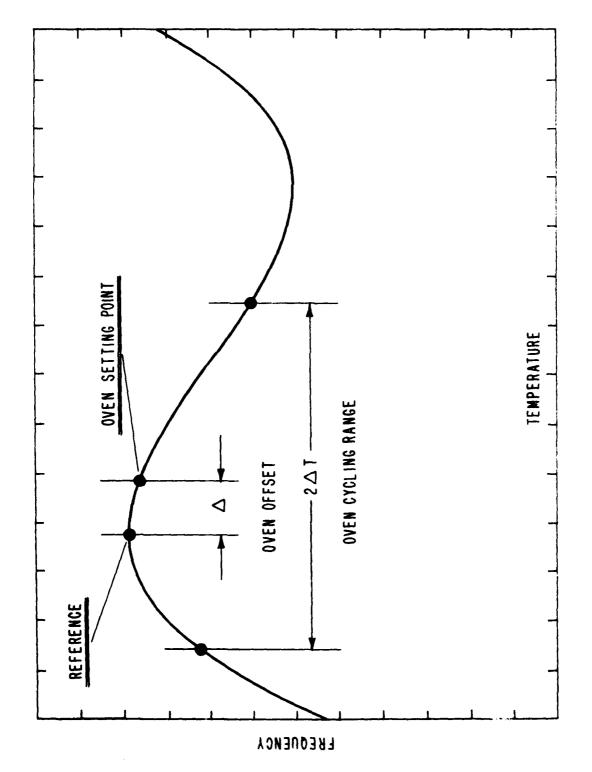


FIGURE 36. Defiritions Relating to Ovenized Oscillators

a measure of the quality of the crystal oven. The result of sinusoidal temperature variations of magnitude AT = 60K about the inflection temperature of the static AT cut f-T curve is shown in Fig. 37. Temperature varies as sin ot; various o values are shown. In Fig. 38 is given the result of variations for $\Delta T = 10K$ about the upper turning point. Figure 39 gives the result when the temperature magnitude is reduced to 5mK. In Fig. 40 the effects of $\Delta T = both \frac{1}{2}$ and 1 mK on the orbit shapes is seen. Figure 41 shows the result of sinusoidal variations in temperature of magnitude \pm_{1} , mK about the turning point of the static f-T curve of Fig. 25. With a period of 8 hours, the frequency change is about 3×10^{-17} ; when the period is lengthened to 1 week, the change is still about 1 x $10^{-1.3}$, whereas the static f-T curve would predict a change of only a few parts in 10^{-1} . irrespective of cycling rate. Since thermal transients, temperature cyclings, and fluctuations cannot be entirely avoided, the dynamic temperature effect is a very important consideration for high precision resonators. The magnitude of the effect can fortunately be reduced by recourse to doubly rotated quartz cuts, specifically the SC-cut. 14,15

AMBIENT VIBRATIONS

Insensitivity to acceleration is one of the most prominent criteria by which to judge high stability resonators. At present, the rule-of-thumb number relating frequency shift to applied acceleration is 2 x 10^{-9} per g, although efforts are underway to reduce this value. (See references given in Reference 20.) It appears that a sensitivity of 1 x $10^{-11}/g$ is a reasonable goal for the next few years. These two sensitivity figures have been used in conjunction with published values for acceleration levels in various environments $^{21-26}$ to produce the frequency shift entries shown in Fig. 42 and Fig. 43.

From Fig. 42 and Fig. 43 it is seen that the decrease in acceleration sensitivity to parts in 10^{11} per g and beyond is highly desirable. The smallest entry in the figures is due to microseisms. $^{22-26}$ Figure 44, taken from Reference 24, shows the very interesting spectrum of microseisms observed at the earth's surface. It appears that the spectrum of displacement versus period plotted on equal log-log scales consists of portions of +2 and -2 slope peaking rather sharply at a period of about 7 seconds. The maximum amplitude of acceleration is approximately 4 micro-g's, and is constant along the +2 slope line. For oscillators with high acceleration sensitivity this source of disturbance would have to be eliminated by mounting supports even for isolated, underground installations. Building vibrations are far greater than microseisms, and their noise spectra must be considered in the installation of precision oscillators. In certain locations, the accelerations due to earthquakes must be allowed for, although the vibrations found in urban environments produce accelerations associated with earthquakes of magnitude 3-4 on the Richter scale.²³ This scale is defined, for shallow shocks, as the log_{10} of the maximum trace amplitude in micrometers with which the standard short-period torsion seismometer would register that earthquake at an epicentral distance of 100km.

Two crystal resonator structures having decreased acceleration sensi-

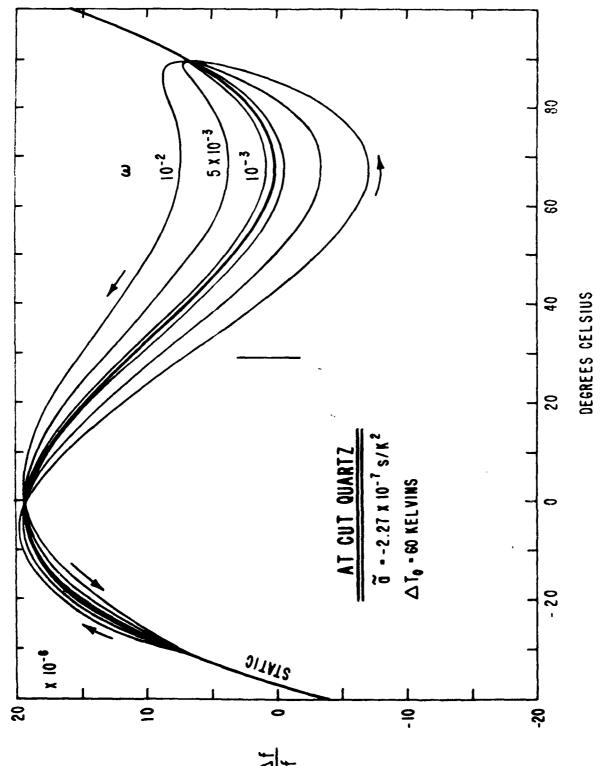


FiGURE 37. Oynamic Thermal Effect; if, = 60K

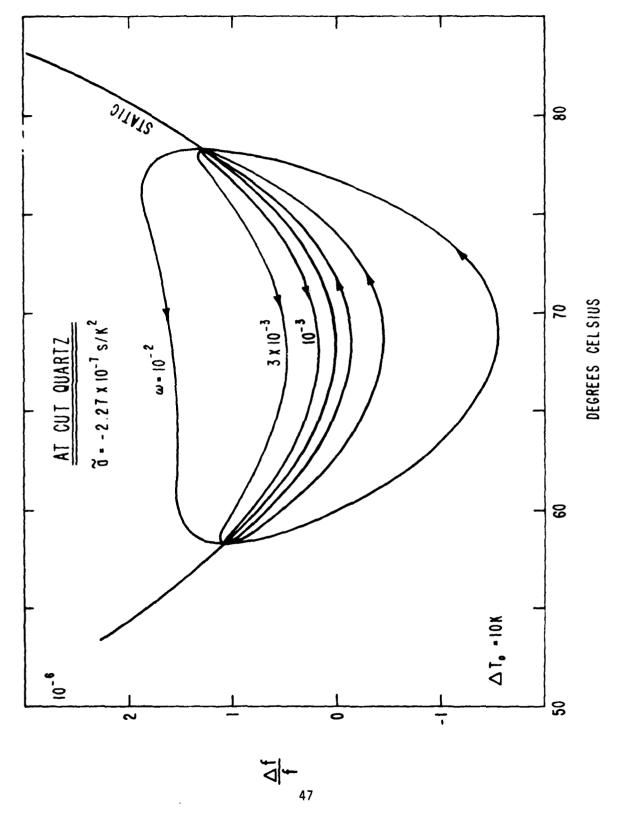
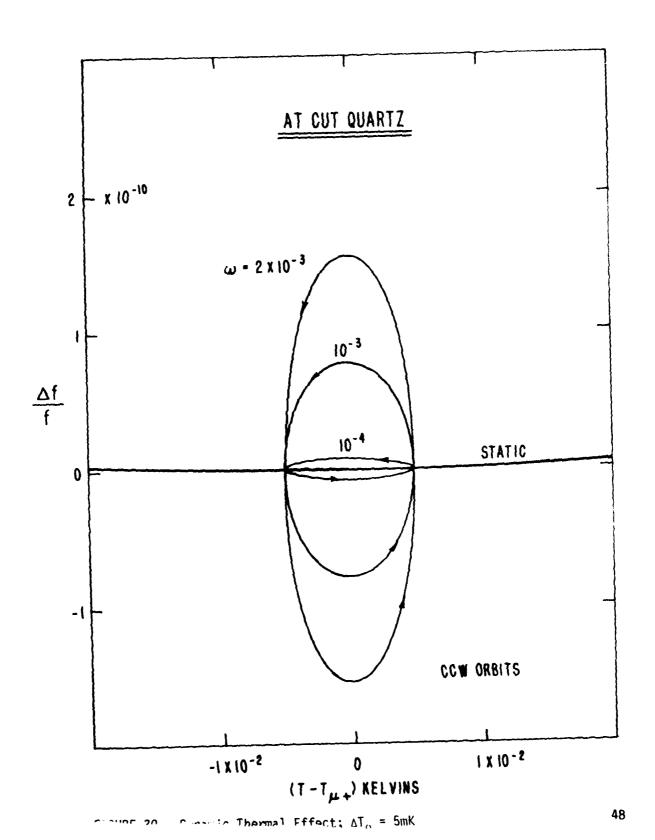


FIGURE 38. Dynamic Thermal Effect; $\Box T \approx 10 \mathrm{K}$



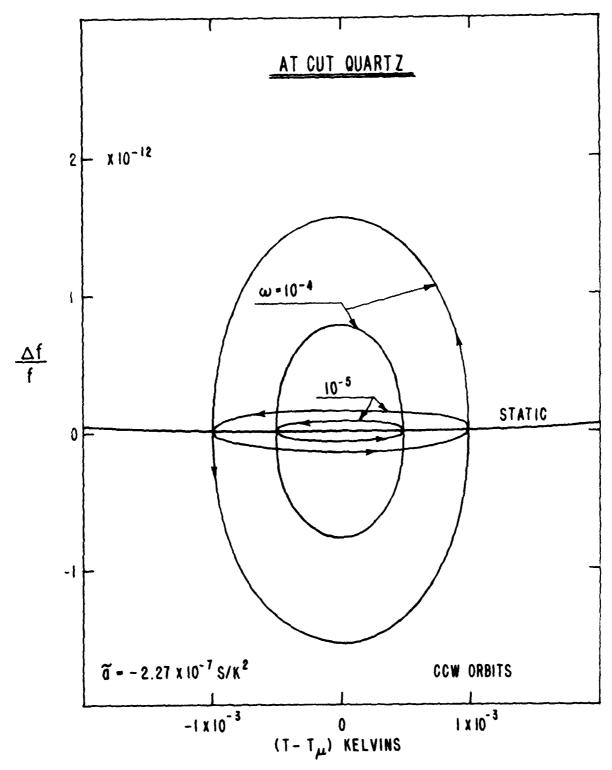


FIGURE 40. Dynamic Thermal Effect: $\Delta T_o = 0.5$ and 1 mK 49

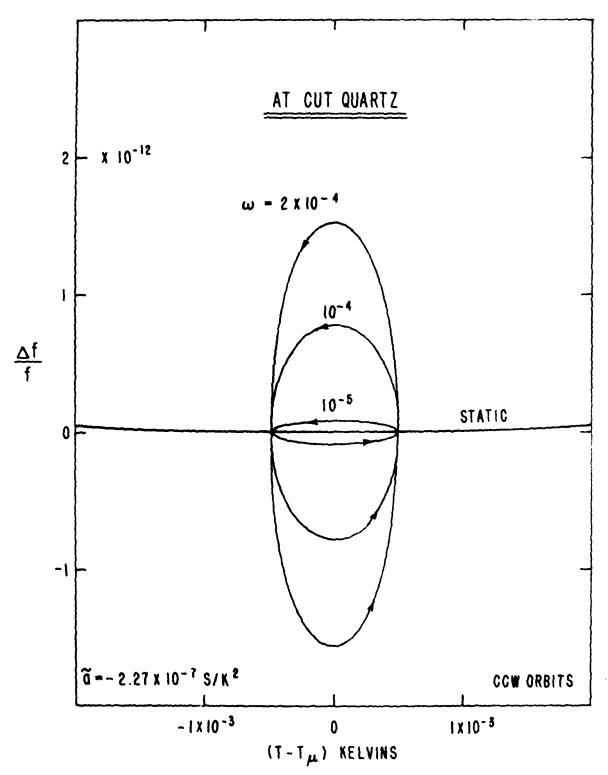


FIGURE 41. Dynamic Thermal Effect; ΔT_{α} = 0.5mK

	ACCEL FRATION	OSCILLATOR SENSITIVITY	SENSITIVITY
ENVIRONMENT	C IINITS	2 x 10 - 9%	3/11 - 01 X I
		FREQUENCY SHIFT	SY SHIFT
ROAD VEHICLES			
TRACTOR-TRAILER (3-80 HZ)	0.2 PEAK	(01-) \$	(21-) 2
PANEL TRUCK	0.02 RMS	4 (-11)	2 (-13)
ARMORED PERSONNEL CARRIER	0.5 TO 3 RMS	(6-) 9	3 (-11)
RAILROADS			
STANDARD	PEAK	2 (-9)	(1-)
SOFT - RIDE	O.I PEAK	2 (-10)	1 (-12)
CROSSING TRACK/SWITCHES	6 PEAK	1.2(-8)	(11-) 9
NORMAL COUPLING & SWITCHES	30 PEAK	(8-) 9	3 (-10)
SHIPS			
NORMAL CALM	0.02 TO 0.8	(6-) 97	8 (-12)
ROUGH SEAS	8.0	(6-) 91	8 (-12)
SLAM & EMERGENCY MANEUVERS	10 3	(6-) 9	3 (-11)

FIGURE 42. Ambient Accelerations: Road/Rail/Ship

	ACCEL FRATION OSCILLATOR SENSITIVITY	OSCILLATOR :	SENSITIVITY
ENVIRONMENT	G IINITS	2 x 10 - %	9/11 - 01 x 1
		FREQUENCY SHIFT	Y SHIFT
AIRCRAFT			
PROPELLER	0.3 TO 5 RMS	1 (-8)	5 (-11)
HELICOPTER	0.1 TO 7 RMS	1.4 (-8)	(11-) 2
JET	0.02 TO 2 RMS	(6-)	2 (-11)
ROCKETS			
MISSILES (BOOST PHASE)	15	3 (-8)	1.5 (-10)
ABLE-I (MULTI-STAGE)	30	(8-) 9	3 (-10)
VENUS ENTRY	310 MAX.	6.2(-7)	3.1(-9)
SEISMIC			
EARTHQUAKE (EL CENTRO, CA, 1940)	0.33 MAX.	(01-)9'9	3,3(-12)
MICROSEISMS	4 x 10 -6 MAX.	8 (-15)	4 (-17)
BUILDINGS (QUIESCENT)	0.02 RMS	4 (-11)	2 (-13)

FIGURE 43. Ambient Accelerations: Aircraft/Rocket/Seismic

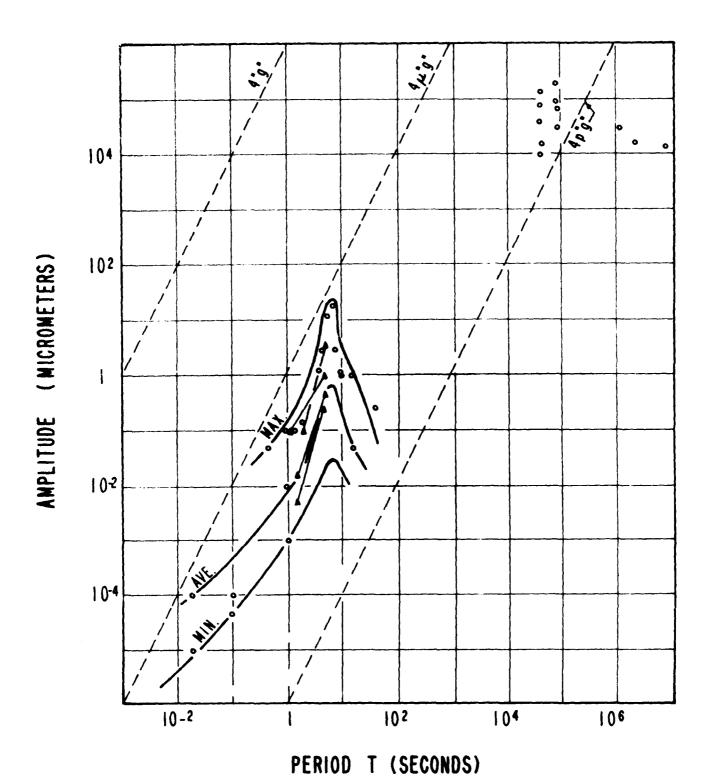


FIGURE 44. SEISMIC NOISE OF THE EARTH'S SURFACE

tivity for accelerations in the thickness direction are the ring-supported and grooved configurations. $2^{0,27}$ Several varieties of these are displayed in Fig. 45 and Fig. 46; Fig. 45 gives the underlying rationale for using ring-supported resonators when the acceleration is applied in the thickness direction. The frequency shift is proportional to displacement of the central membrane, and because of the double cantilever edge conditions this displacement is smaller than in conventional simply supported resonators. Figure 47 defines the parameters pertinent to the geometry of the plano-convex ring-supported resonator. These parameters are listed below for resonator B-4: (in mm)

$$\phi_o = 7.950;$$
 $\phi_i = 6.020;$ $\phi_e = 4.064;$
 $t_o = 1.000;$ $t_i = 0.662;$ $t_c = 0.700;$
 $R = 120.$

The measured frequency constant was N=1.770~MHz-mm. In Fig. 48 is shown the experimental frequency shift versus acceleration for resonator I-4, an AI-cut, plano-convex, ring-supported resonator with acceleration applied along the X axis. The frequency shifts are seen to saturate at high g levels, leading to the conclusion that the acceleration-frequency sensitivity coefficient is nonlinear. Figure 49 gives the corresponding curves for resonator B-4. The concavo-convex ring-supported geometry in Fig. 46 is a preferred structure for unidirectional acceleration loads directed toward the radii of curvature. The resonator center is thicker than the junction to the ring to permit energy trapping.

ACOUSTIC VISCOSITY

One of the foremost reasons for using quartz as a vibrator material is its extremely low acoustic attenuation, as manifested in the large values of quality factor (Q) measured, typically as much as 28

$$Q \approx 10^7 / f_0,$$
 (20)

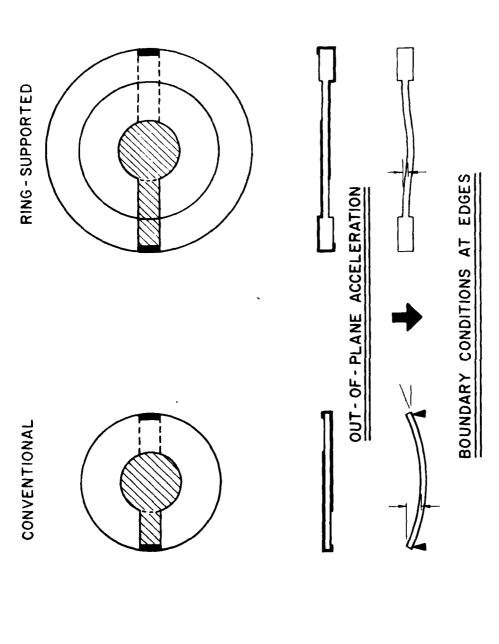
at room temperature, where f is the operating frequency in MHz. This is shown in Fig. 50. The values in (20) are met, and exceeded, only by very careful design of the mounting structure at frequencies below a few tens of MHz²⁸; above these frequencies the intrinsic internal friction of the material predominates. Calculation of the intrinsic loss follows from the measurement of the elements η_{ij} of the acoustic viscosity tensor.²⁹

For the rotated-Y-cuts of quartz, the piezoelectrically driven shear mode has an effective viscosity of

$$\eta' = \eta_{66} \cos^2\theta + \eta_{44} \sin^2\theta + 2\eta_{14} \sin\theta \cos\theta.$$
 (21)

Substituting the elastic stiffness c_{ij} for the viscosities in (21) yields

CRYSTAL RESONATORS



SIMPLY SUPPORTED:

(a) EDGE FORCES; ZERO DISPLACEMENTS

(b) NO EDGE TORQUES, NONZERO SLOPES

DOUBLE CANTILEVER:

(a) EDGE FORCES, ZERO DISPLACEMENTS

(b) EDGE TORQUES, ZERO SLOPES

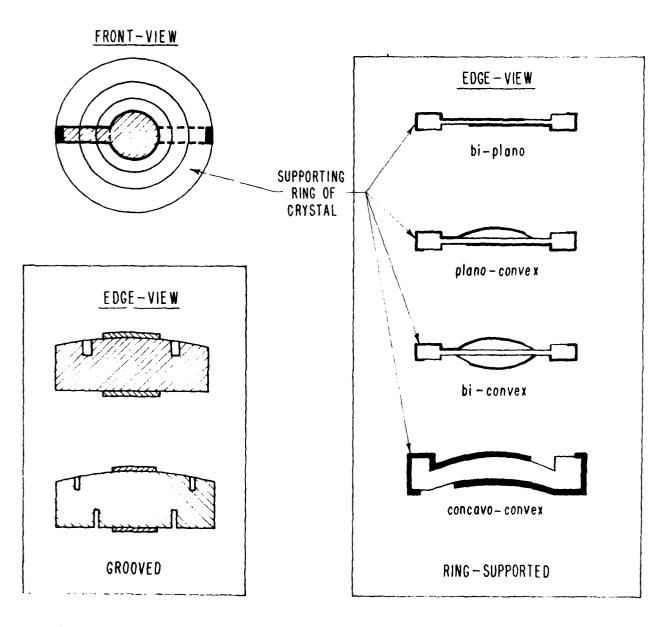


FIGURE 46. Ring-Supported and Grooved Crystal Resonators

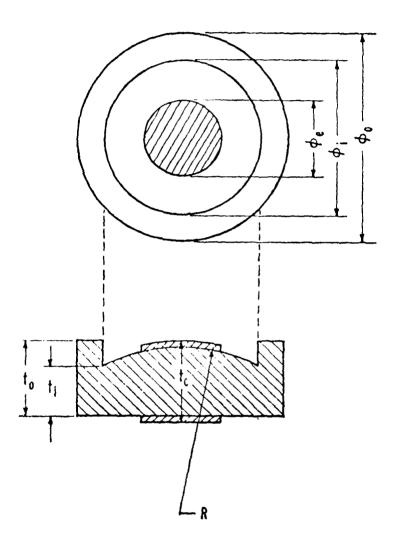
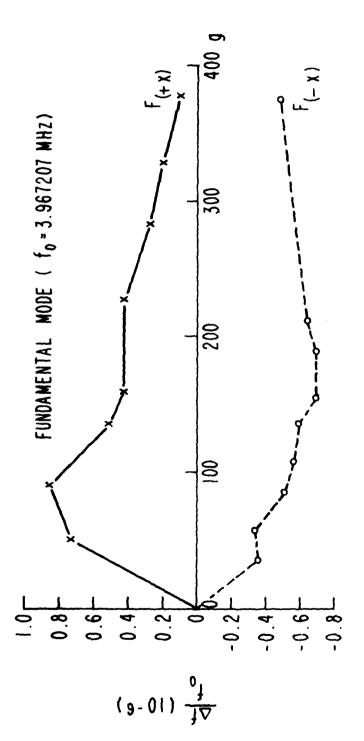


FIGURE 47. Definitions of Ring-Supported Resonator Parameters



ACCELERATION FREQUENCY EFFECTS OF X-AXIS DIRECTION FOR SAMPLE NO. T-4 FIGURE 48.

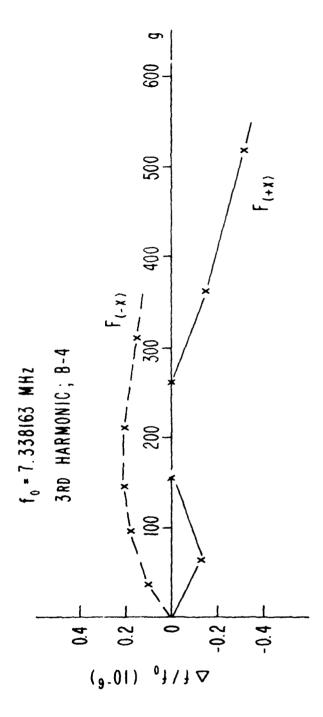


FIGURE 49.
ACCELERATION - FREQUENCY EFFECTS OF X-AXIS DIRECTION FOR SAMPLE NO. 8-4

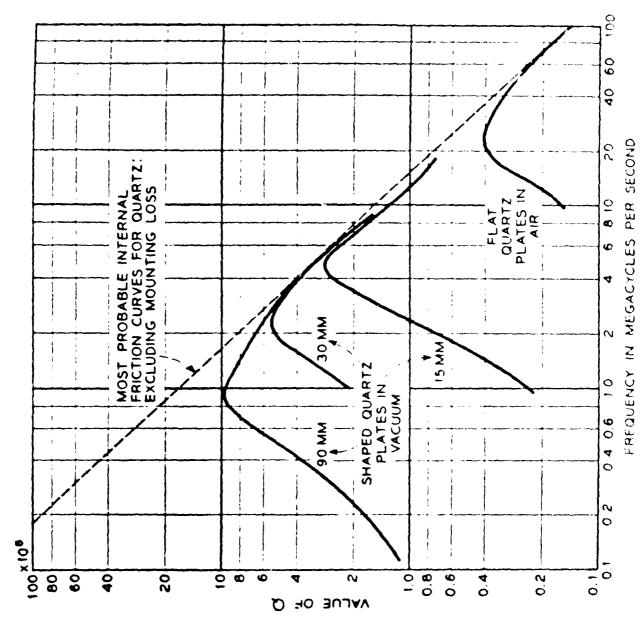


FIGURE 50. Experimental alwes of Q versus Freduency for Swartz

the corresponding effective stiffness of for this pure mode. Then the motional time constant \cdot , representing the intrinsic losses is 36.41

$$c_i = r_i \cdot /c^*. \tag{22}$$

Since .. is also equal to

$$R_1 = R_1 - C_1 = 1/(U_0, Q),$$
 (23)

both the intrinsic () at an operating frequency $f_{ij}=\omega_{ij}/2\pi$ and the motional resistance R_{ij} of the equivalent electrical network may be computed, as the motional capacitance is separately calculated.

When mounting losses cannot be neglected, it is found that these losses depend upon the plate diameter-to-thickness ratio τ in the following manner:

$$\tau_1(z) = \tau_1 - (1 + (z/z_0)^2)$$
, (24)

where : is the intrinsic value of τ_1 , and depends on the material used, being 10 to 15 femtoseconds for room temperature quartz, and ζ_0 is a function of the resonator design, with ζ_0 = 60 a good average. This is demonstrated by Fig. 51 and Fig. 52, where the data of Fig. 50 and additional data are plotted against ζ . It is seen that for $\zeta < \zeta$ the loss, as reflected in τ_1 is governed by a curve with -2 slope; beyond ζ_0 , the curve is constant, with zero slope. The calculation of τ_1 has been extended to the general doubly rotated thickness mode case. 14

COSMIC RAYS

Cosmic rays, and other unshieldable radiation sources, produce cumulative contributions to long-term aging. Knowing the radiation sensitivity of natural and cultured quartz under a variety of treatments permits an evaluation of resonator performance due to this source.

The raw data are contained in Fig. 53, Fig. 54 and Fig 55. Using the values for the photon flux given and a number of simplifying assumptions, the frequency shift due to this source is given in Fig. 56 for two values of quartz sensitivity. The unit "rad" is defined as "The standard unit of absorbed dose, equal to energy absorption of 100 ergs per gram (0.01 joules per kilogram)."⁴⁰ The assumptions used are (a) unprotected crystal, (b) total incident energy is absorbed, and (c) sea level flux. Ip is the photon flux in photons / cm² - sec. The fractional frequency shift $\Delta f/f$ is specified as 10^{-5} ; the time $\tau(s)$ for this shift to occur is

$$\tau(s) \approx 5 \times 10^{12-S}$$
 years. (25)

Similarly for neutrons, assumptions (a) and (b) listed above, plus the sensitivity value $\Delta f/f \sim 10^{-21}$ cm²/n. lead to the order-of magnitude figures given in Fig. 57 for three values of altitude. $\mathcal F$ n is the neutron

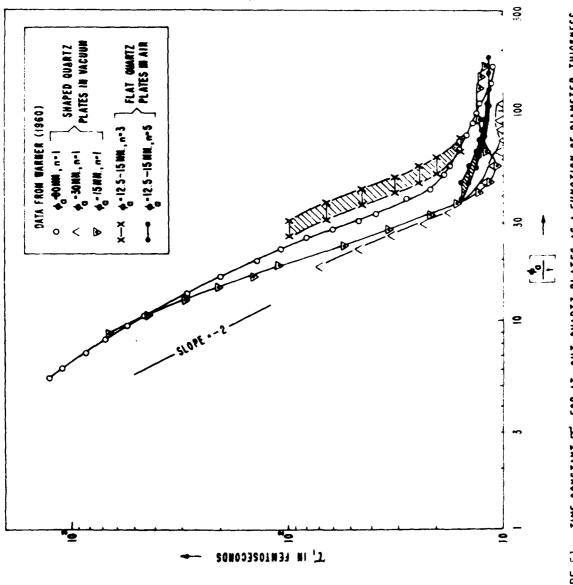
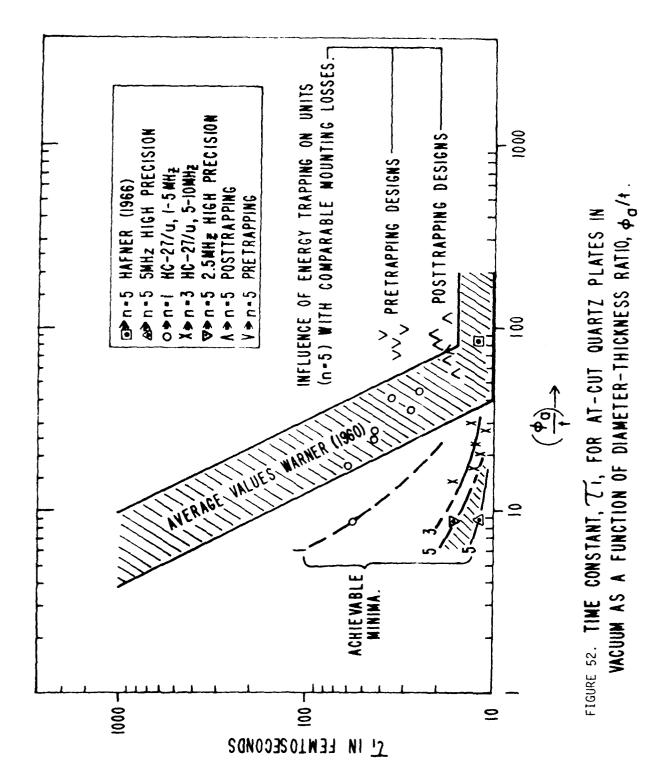


FIGURE 51. TIME CONSTANT, T, FOR AT-CUT QUARTZ PLATES AS A FUNCTION OF DIAMETER-THICKNESS RATIO, & AVERAGE VALUES

And the second of the second o



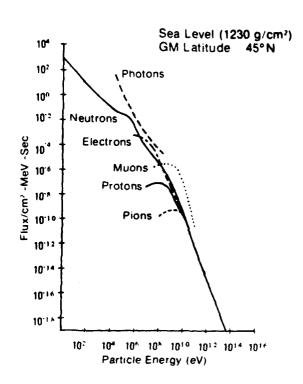
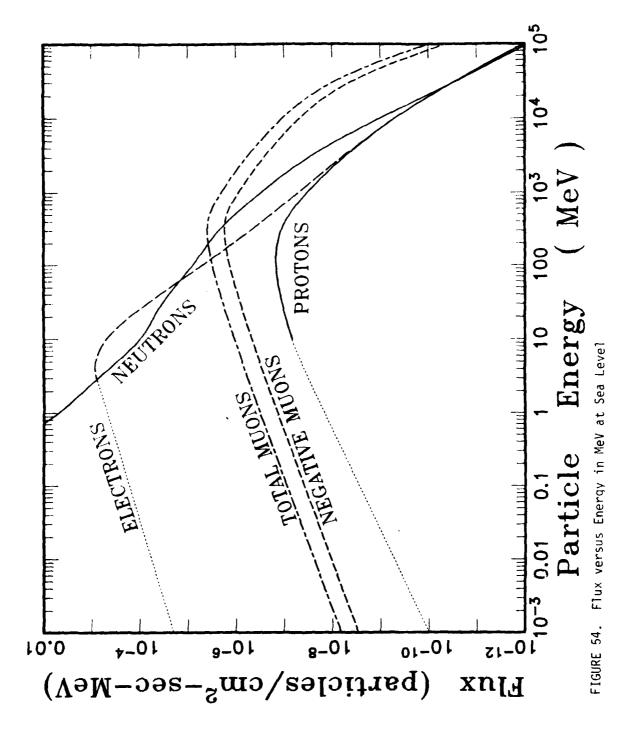


FIGURE 53. Cosmic Ray Flux versus Particle Energy at Sea Level



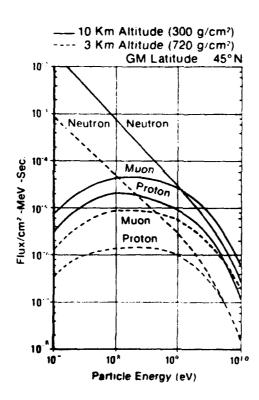


FIGURE 55. Flux versus Energy at 3 and 10 km Height

Specification: $\Delta F/F = 10^{-5}$

Time for specified shift: \mathbb{C} (s) $\simeq 5*10^{12}$ -s years

	NATURAL QUARTZ	SWEPT CULTURED
AF/F	$\sim 10^{-11}/{ m RAD}$	$\sim 10^{-12}/{ m RAD}$
d to	~ 1	~ 1
$\mathcal{L}^{(12)}$	\sim 5 YRS.	\sim 50 YRS.
(hI)2	~ 5 HOURS!	\sim 0.5 YRS.
/ L CLLC		

NOTES: 1/ UNPROTECTED CRYSTAL

2/ TOTAL ENERGY ABSORBED

3/ SEA LEVEL

4/ F IN PHOTONS/CM2-SEC

FIGURE 56. Frequency Shift for Photons

Specification: $\Delta F/F = 10^{-S}$

TIME FOR SPECIFIED SHIFT: $\mathcal{T}(s) \simeq 10^{14} - s/4f_{N}$ YEARS

	SEA LEVEL	3 KM	10 KM
J N	$\sim 10^{-3}$	$\sim 10^{-2}$	9'0 ~
J (12)	$\sim 25,000$ yrs,	\sim 2,500 yrs.	\sim 4 YRS.
C (14)	~ 250 yrs,	\sim 25 yrs.	\sim 16 days!

NOTES: 1/ UNPROTECTED CRYSTAL

2/ TOTAL ENERGY ABSORBED

 $3/~^{\Delta}\text{F/F} \sim 10^{-21}~\text{cm}^2/\text{Neutron}$

4/ 3 N IN NEUTRON-SEC/CM2

FIGURE 57. Frequency Shift for Neutrons

flux in neutron-sec/cm². The fractional frequency shift is specified as 10^{-5} , and the time $\tau(s)$ to reach this shift is

$$\tau(s) = 10^{14-s} / 4 \mathcal{F}_n \text{ years.}$$
 (26)

CONCLUSION

Over the years, the stabilities of precision quartz oscillators have improved according to a simple law. With frequency stability written as $\Delta f/f = 10^{-5}$, s is given by s = 6 + 0.1(Y-X); Y is the year date and x = 1920 for large-scale production models; this is seen in Fig. 58. Laboratory oscillators, therefore, can be projected to have stabilities in the 10^{-14} range in the time frame 2000 if the current rate of improvement is maintained. There do not appear to be intrinsic impediments to achieving 10^{-14} stabilities with quartz elements by 2000, but in order to do so the various "subtle" effects affecting stability will have to be subjected to increasingly fine scrutiny. Figure 59 gives a broad view of technology progression for crystal resonators from 1945 onward. The most interesting developments are yet to come.

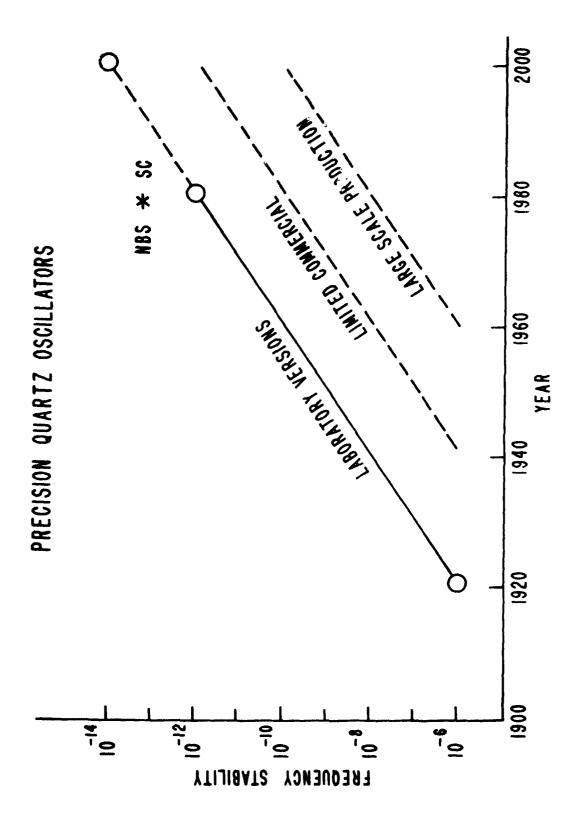


FIGURE 58. Frequency Stability versus Year

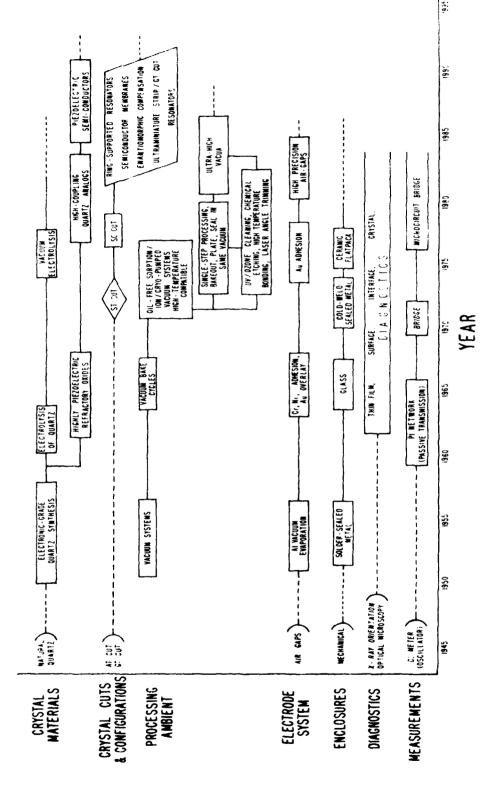


FIGURE 59. Progression in Crystal Technology

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